# (12) STANDARD PATENT

(11) Application No. AU 2016262984 B2

## (19) AUSTRALIAN PATENT OFFICE

(54) Title

Adjustable ballast bulb for a sailing vessel

(51) International Patent Classification(s)

**B63B 41/00** (2006.01) **F16B 21/12** (2006.01) **B63B 3/38** (2006.01) **F16H 55/02** (2006.01)

- (21) Application No: **2016262984** (22) Date of Filing: **2016.05.16**
- (87) WIPO No: WO16/185357
- (30) Priority Data
- (31) Number (32) Date (33) Country **2015/03491 2015.05.19 ZA**
- (43) Publication Date: **2016.11.24** (44) Accepted Journal Date: **2020.10.01**
- (71) Applicant(s) Sarah-May Steenkamp
- (72) Inventor(s)
  Cronwright, Keith Louis Featherstone
- (74) Agent / Attorney
  Michael Buck IP, PO Box 78, Red Hill, QLD, 4059, AU
- (56) Related Art
  US 7938076 B2
  US 4686922 A
  WO 2007104086 A1
  GR 1006319 B

#### (12) INTERNATIONAL APPLICATION PUBLISHED UNDER THE PATENT COOPERATION TREATY (PCT)

### (19) World Intellectual Property Organization

International Bureau





(10) International Publication Number WO 2016/185357 A2

(43) International Publication Date 24 November 2016 (24.11.2016)

24 November 2016 (24.11.2016)

(51) International Patent Classification: Not classified

(21) International Application Number:

PCT/IB2016/052815

(22) International Filing Date:

16 May 2016 (16.05.2016)

(25) Filing Language:

English

(26) Publication Language:

English

(30) Priority Data:

2015/03491

19 May 2015 (19.05.2015)

ZA

- (72) Inventor; and
- (71) Applicant: CRONWRIGHT, Keith Louis Featherstone [ZA/ZA]; 25 Baviaanskloof Estate, Baviaanskloof Road, Hout Bay, Cape Town (ZA).
- (74) Agent: CREASE, Vanessa; Brian Bacon Inc., 2nd Floor, Mariendahl House, Newlands on Main, Main Road, Newlands 7700, Cape Town (ZA).
- (81) Designated States (unless otherwise indicated, for every kind of national protection available): AE, AG, AL, AM,

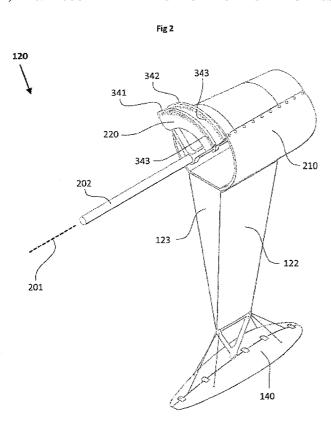
AO, AT, AU, AZ, BA, BB, BG, BH, BN, BR, BW, BY, BZ, CA, CH, CL, CN, CO, CR, CU, CZ, DE, DK, DM, DO, DZ, EC, EE, EG, ES, FI, GB, GD, GE, GH, GM, GT, HN, HR, HU, ID, IL, IN, IR, IS, JP, KE, KG, KN, KP, KR, KZ, LA, LC, LK, LR, LS, LU, LY, MA, MD, ME, MG, MK, MN, MW, MX, MY, MZ, NA, NG, NI, NO, NZ, OM, PA, PE, PG, PH, PL, PT, QA, RO, RS, RU, RW, SA, SC, SD, SE, SG, SK, SL, SM, ST, SV, SY, TH, TJ, TM, TN, TR, TT, TZ, UA, UG, US, UZ, VC, VN, ZA, ZM, ZW.

(84) Designated States (unless otherwise indicated, for every kind of regional protection available): ARIPO (BW, GH, GM, KE, LR, LS, MW, MZ, NA, RW, SD, SL, ST, SZ, TZ, UG, ZM, ZW), Eurasian (AM, AZ, BY, KG, KZ, RU, TJ, TM), European (AL, AT, BE, BG, CH, CY, CZ, DE, DK, EE, ES, FI, FR, GB, GR, HR, HU, IE, IS, IT, LT, LU, LV, MC, MK, MT, NL, NO, PL, PT, RO, RS, SE, SI, SK, SM, TR), OAPI (BF, BJ, CF, CG, CI, CM, GA, GN, GQ, GW, KM, ML, MR, NE, SN, TD, TG).

#### Published:

 without international search report and to be republished upon receipt of that report (Rule 48.2(g))

#### (54) Title: ADJUSTABLE BALLAST BULB FOR A SAILING VESSEL



(57) Abstract: A sailing vessel is disclosed which comprises a hull, a keel depending from the hull and a mast. A ballast bulb is provided at the lower end of the keel. A first control mechanisms is provided for rotating the ballast bulb about a transverse axis to change the angle of attack of the bulb. A second control mechanism is provided for rotating the bulb about a longitudinal axis of the vessel.



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#### ADJUSTABLE BALLAST BULB FOR A SAILING VESSEL

#### **FIELD OF THE INVENTION**

This invention relates to a monohull sailing vessel with sailing vessel apparatus. In particular, it relates to an adjustable ballast bulb for a monohull sailing vessel.

#### **BACKGROUND TO THE INVENTION**

Any references to methods, apparatus or documents of the prior art are not to be taken as constituting any evidence or admission that they formed, or form part of the common general knowledge.

Over time sailing vessels' hull design has evolved from 'displacement' hulls with a fixed maximum boat speed consequent on hull being 'pushed' through the water to planing hulls. Planing hulls achieve significantly higher forward velocity by sailing atop or 'planing' on water thereby reducing resistance or drag. Consequent on this increased boat speed was the realisation by naval architects that a keeled sailboat's speed could exceed the wind's velocity. This phenomenon only occurs when vessel is sailed 'off the wind' (as in vessel is aligned at greater than 90° to the direction of the wind). This 'off the wind' attitude is in contrast to a 'close-hauled' attitude, wherein vessel is aligned at less than 90° to the wind's direction.

A further consequence of this ability to exceed wind speed when sailing 'off the wind' being that such vessel would therefore never sail directly downwind as their sails would collapse when vessel's speed attains wind speed (as in 'running' with wind -vessel orientated 180° to wind's direction). Modern sailing vessels therefore always sail either 'off the wind' or 'close-hauled'.

Both these attitudes result in the vessel's hull 'heeling' secondary to the force generated by the wind on the sails. Control of vessel's 'heeling' moment is therefore required.

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#### **SUMMARY OF THE INVENTION**

According to the present invention there is provided a sailing vessel having a hull, a keel, a mast, a ballast bulb in the form of a hydrofoil providing lift and ballast at the base of the keel and a control apparatus for controlling adjustment of the ballast bulb in relation to the keel, wherein the control apparatus includes: a first rotating mechanism for rotating the ballast bulb around a lateral axis of the ballast bulb to change the pitch of the ballast bulb in relation to the keel; and a second rotating mechanism for rotating the ballast bulb around a longitudinal axis of the ballast bulb to change the roll of the ballast bulb in relation to the keel, and change the direction of lift generated by the hydrofoil.

0. Said control apparatus can include a pitch actuator and a roll actuator connected to the ballast bulb by means of wires between each quadrant of an upper surface of the ballast bulb and the keel.

The ballast bulb preferably includes a compartment extending longitudinally within the ballast bulb and a movable mass housed within the compartment and movable longitudinally in the compartment to alter the ballast distribution in the ballast bulb. There can be at least one further compartment in the ballast bulb which is accessible when the sailing vessel is out of the water.

In an aspect, the present invention provides a sailing vessel having a hull, a keel, a mast, a ballast bulb providing ballast at the base of the keel and a control apparatus for controlling adjustment of the ballast bulb in relation to the keel, wherein the control apparatus includes: a first rotating mechanism for rotating the ballast bulb around a lateral axis of the ballast bulb to change the pitch of the ballast bulb in relation to the keel; and a second rotating mechanism for rotating the ballast bulb around a longitudinal axis of the ballast bulb to change the roll of the ballast bulb in relation to the keel, wherein said control apparatus includes a pitch actuator and a roll actuator connected to the ballast bulb by means of wires between each quadrant of an upper surface of the ballast bulb and the keel.

In an embodiment, the ballast bulb includes a compartment extending longitudinally within the ballast bulb and a movable mass housed within the compartment and movable longitudinally in the compartment to alter the ballast distribution in the ballast bulb.

In another embodiment, the ballast bulb includes at least one further compartment which is accessible when the sailing vessel is out of the water.

#### **BRIEF DESCRIPTION OF THE DRAWINGS**

The invention will now be described, by way of example only, with reference to the accompanying representations in which:

0. Figure 1 A is a perspective view from the rear of a sailing vessel in accordance with a first embodiment of the present invention;

Figure 1B is an enlarged view of Figure 1A;

Figure 1C is a perspective view from the front of the sailing vessel of Figure 1A;

Figure 1D is a rear view of the sailing vessel of Figure 1A showing the keel in a canted position;

т2 Figure 1E is a perspective view from the rear of a sailing vessel in accordance with a second embodiment of the present invention having an alternative rigging arrangement;

Figure 1F is the view of Figure 1E showing a mainsail;

Figure 1G is an enlarged view of Figure 1E;

Figure 1H is a further enlarged view of Figure 1 E showing the rear of the sailing vessel;

20 Figure 1I is a perspective view from the front of the sailing vessel of Figure 1E;

Figure 1J is a side view of the sailing vessel of Figure 1E;

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Figure 2 is a perspective view of a keel arrangement in accordance with aspects of the present invention;

Figure 3A is a cross-sectional view of a keel mechanical canting mechanism in accordance with an aspect of the present invention;

Figure 3B is a perspective view of the keel mechanical canting mechanism of Figure 3A;

Figure 3C is a further perspective view of the keel mechanical canting mechanism of Figure 3A;

Figure 4A is a cross-sectional view a mast mechanical canting mechanism in accordance with an aspect of the present invention;

Figure 4B is a cut-away perspective view of the mast mechanical canting mechanism of Figure 4A;

0. Figure 4C is a further perspective view of the mast mechanical canting mechanism of Figure 4A;

Figure 4D is another front perspective view of the mast mechanical canting mechanism of Figure 4A;

Figure 4E is a perspective view of the mast mechanical canting mechanism in isolation from the sailing vessel with the first embodiment of the rigging arrangement;

Figure 4F is a perspective view of the mast mechanical canting mechanism in insolation from the sailing vessel with the second embodiment of the rigging arrangement;

Figure 5A is a perspective view of a keel arrangement in accordance with aspects of the present invention;

Figure 5B is a side view of the keel arrangement of Figure 5A;

Figure 5C is a side view of an alternative embodiment of the keel arrangement;

20 Figure 6A is a perspective view from the front of a generally cylindrical or truncated conical housing of a keel arrangement in accordance with an aspect of the present invention;

Figure 6B is a perspective view from the front of the generally cylindrical or truncated conical housing of Figure 6A;

Figure 6C is front view of the generally cylindrical or truncated conical housing of Figure 6A, wherein keel is canted at 450;

Figure 7A is a front view of a ballast bulb in a first rotated position of a roll rotation in accordance with an aspect of the present invention;

Figure 7B is a front view of the ballast bulb of Figure 7A in a second rotated position of a roll rotation;

Figure 7C is a side view of the ballast bulb of Figure 7A in a first rotated position of a pitch rotation in accordance with an aspect of the present invention;

.0 Figure 7D is a side view of the ballast bulb of Figure 7A in a second rotated position of a pitch rotation;

Figure 7E is a perspective view of the axes of rotation of the ballast bulb of Figure 7A;

Figure 7F is a cut-away perspective view of the axes of rotation of the ballast bulb of Figure 7A;

Figure 8A is a plan view of a control mechanism of the ballast bulb of Figure 7A;

Figure 8B is a perspective view of the control mechanism of Figure 8A;

15 Figure 9A is a perspective view of an open ballast bulb in accordance with an aspect of the present invention;

Figure 9B is a perspective view of a closed ballast bulb in accordance with an aspect of the present invention;

Figure 10A is a side view of a propulsion system in accordance with an aspect of the present invention;

20 Figure 10B is a perspective view of the propulsion system of Figure 10A;

Figure 11 A is an underside view of a hull in accordance with an aspect of the present invention;

Figure 11 B is a side view of the hull of Figure 11A;

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Figure 12A is a perspective cut-away view of a hull of a sailing vessel showing additional aspects of the present invention;

Figure 12B is a perspective view of a helmsman's pedestal in accordance with an aspect of the present invention; and

Figure 12C is a perspective view of a deck seat in accordance with an aspect of the present invention.

#### DETAILED DESCRIPTION WITH REFERENCE TO THE DRAWINGS

A sailing vessel, various sailing vessel apparatus and additional aspects are described herein. It should be appreciated that some of the apparatus and aspects may be used in combination or as stand-alone apparatus.

Referring to Figures 1A to 1D, a sailing vessel (100) is shown. The sailing vessel (100) may be of any practical size, which may be powered in part or fully by wind, including a mega-yacht, super-yacht, medium yacht, small-yacht, dinghy sailing yacht, or a scaled model sailing vessel with remote control. Figures 1 E to 1 J show a sailing vessel (100) with a rigging arrangement including a backstay arrangement which is described further with respect to Figure 4F.

The sailing vessel (100) includes a ballast bulb (140) which may be articulated. The ballast bulb (140) and its associated features is described with reference to Figures 7A to 7F, Figures 8A and 8B, and Figures 9A and 9B. Additional features of a sailing vessel (100) are described herein which may optionally be used in combination with the described adjustable ballast bulb (140). However, it should be understood that an adjustable ballast bulb (140) as described may be used with other sailing vessels of varying design.

The adjustable ballast bulb (140) may be used with a sailing vessel with a non-canting keel, or with a sailing vessel including hydraulic canting keels or mechanical canting keels. A particular embodiment of a mechanical canting keel is described as an example embodiment.

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The ballast bulb may be "hydrofoiled" in that the alignment and 'angle-of-attack' are adjustable. The ballast bulb may be rotatable around a long axis and inclined to its horizontal plane variable. Rotation around long axis allows an operator to direct the vector of force generated by foil's forward movement through water, between either raising hull superiorly (lifting out water) and/or as 'righting moment'. The quantity of this "lift" (or force) may be adjusted by the operator increasing or decreasing foil's 'angle-of-attack'.

The sailing vessel (100) may have a hull (110) which is hard-chined with a port semi-hull chine (111) and a starboard semi-hull chine (112) to provide a bi-planer-type configuration of a monohull. Sheer strakes (113, 114) substantially perpendicular to the port and starboard semi-hull chines (111, 112) may be provided respectively. This is an example embodiment of a hull (110) form and other forms may also be provided.

Additional to the concept are multiple adjustable devices that function collectively to maintain either port or starboard semi-hull of the bi-planer monohull on an even-keel.

Control of the vessel's 'heeling' moment may be provided via multiple adjustable devices. Adjustments may be made such that vessel's hull aims to remain on an even-keel at all times. One of these forms of control is provided by the adjustable ballast bulb (140).

A component of the design is the reconfiguration of the monohull sailing vessel's hull design such that a unitary hull incorporates two distinct / separate planing 'semi-hulls', port and starboard. When sailed, the sailing vessel's orientation to the water is such that either port or starboard semi-hull is maintained on an even-keel parallel to the water's surface. This is achieved by multiple controls both traditional and as described. The adjustments thereto, countering the 'heeling moment' of the hull secondary to wind pressure on the sails. Traditional control of a sailing vessel's heeling moment is achieved through adjustments made to sails, ballast and steerage. This description includes additional apparatus for adjusting righting moment.

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The sailing vessel (100) may have a keel arrangement (120) which cants around an axis of rotation running longitudinally fore and aft of the sailing vessel (100).

Figure 1D shows the sailing vessel (100) including a keel arrangement (120) shown in a canted position. This figure also shows that the sailing vessel (100) may include conventional port and starboard rudders (116, 117) which may be retractable depending on the canted position of the keel arrangement (120). The adjustable ballast bulb (140) is also shown in this figure in a rotated position illustrating the effect of the adjustability of the ballast bulb (140).

The keel arrangement (120) is canted by a keel mechanical canting mechanism which is described further below with reference to Figures 2, 3A, 3B and 3C.

.0 The keel arrangement (120) may be configured as a triangle. The base of triangle articulates superiorly with a canting mechanism and the apex of the triangle inferiorly with a ballast bulb hydrofoil. The canting mechanism articulates with and rotates around a shaft located in a semi-circular recess of vessel's bilge. Port and starboard keel-fins constitute the arms of triangle.

The described canting keel arrangement is powered by a linear worm-gear actuator which has the advantage when compared to a hydraulic-powered canting keel, that a worm-gear actuator has high static loading capacity. The worm-gear actuator has a low holding load, which is the force applied to the actuator when not in motion. The consequence of a high static loading capacity is that, once the keel arrangement is positioned in a canted position, it requires no energy to maintain this attitude. The worm of the worm-gear actuator must rotate to move the keel arrangement and in this regard it can be considered self-locking.

The sailing vessel (100) includes a mast (150) which may also be able to cant in relation to the hull (110). A mast mechanical canting mechanism may be provided which is described further below with reference to Figures 4A to 4F.

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A first embodiment of an arrangement of the rigging of the sailing vessel (100) may be provided with an arrangement of two triangular structures shown in Figure 4E. The first triangular structure may be formed of the mast (150), forestay (151) and an extended bowsprit (152) which may be rigid or semirigid. The second triangular structure may be formed of the mast (150) and the port and starboard triangular stays (153, 154) which are formed of rigid rods. An arc of rotation of approximately 40 degrees may be provided. The rigid rods may be of profiled solid metal, round bar or tubular sections. A second embodiment of an arrangement of the rigging of the sailing vessel (100) may be provided with an arrangement of two triangular structures and an additional backstay arrangement shown in Figure 4F. A topmast rigid triangular structure is provided comprising port and starboard topmast sprits (161, 162) and a topmast spar (163). Port (171) and starboard (172) backstays constitute one contiguous cable connected by a backstay link (173) thereby accommodating the canting mast (150) varying the lengths and attitude. Backstays (171, 172) are coupled to port (174) and starboard (175) retractable stern-sprits mounted on the transom. With reference to Figure 1H, port (176) and starboard (177) struts are demountable with locking pins to permit deployment or retraction of the stern sprits (174, 175).

The backstay arrangement shown in Figure 4F has the purpose of avoiding a potential weakness in a mast where shrouds attach. Known designs may have no fixed rigging superior to attachment. Consequently the force directed forward on a mast superior to this point may be at risk of failure due to human error secondary to mal-adjustment of (non-fixed) 'running' backstays. Figure 4F, includes a fixed aft stay arrangement eliminating risk of human error.

In the described embodiment, the boom (155) extends beyond the transom thereby accommodating a larger mainsail. A fixed stay arrangement incorporating topmast (161, 162) and retractable stern sprits (174, 175) permit unimpeded rotation of boom (155). The varying length of the back stays secondary to canting the mast (150) is provided for by the port and starboard backstays (171, 172) forming a contiguous closed loop arrangement via backstay link (173).

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The described sailing vessel has a fixed bowsprit (152). Stern sprits (174, 175) retract prior to the sailing vessel maneuvering astern into a mooring.

The keel arrangement (120) may be formed of a rotational housing and two keel fins (122, 123) which converge to an apex at which a ballast bulb (140) may be provided. The form of the rotational housing is described further below with reference to Figures 5A, 5B, 5C, 6A, 6B and 6C.

Referring to Figure 2, an embodiment of the keel arrangement (120) is shown in isolated detail from the sailing vessel. The keel arrangement (120) may include a rotational housing (210) and two keel fins (122, 123) which converge to an apex at which a ballast bulb (140) may be provided.

The keel arrangement (120) cants around a shaft (202) with an axis of rotation (201) running longitudinally fore and aft of the sailing vessel. The keel arrangement (120) is canted by a keel mechanical canting mechanism (220) part of which is shown in Figure 2 and which is described in more detail in relation to Figures 3A, 3B and 3C.

Referring to Figures 3A, 3B and 3C an embodiment of the keel mechanical canting mechanism (220) is shown. The keel mechanical canting mechanism (220) may include a first worm gear (310) and a first worm (320). The first worm gear (310) may rotate in a fixed relationship with the keel arrangement (120) around the axis (201) of rotation. The first worm (320) may have a helical thread (321) which engages with the teeth (311) of the first worm gear (310).

In one embodiment, the first worm (320) may be a double-enveloping worm for high power transmission which has a waisted-shape which conforms to the arc of the first worm gear (310). A double-enveloping worm provides improved locking as all of the teeth (311) of the first worm gear (310) are in contact with the helical thread (321) of the first worm (320). It should be understood that the first worm and first worm gear may optionally be recirculating ball worm and worm gear in which the threads are filled with bearing balls that recirculate through the gear and worm as it turns, reducing friction and wear in the gear.

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The first worm (320) may be driven by a first driving means (330) which may take the form of two motors or other drive means at either end of a rotational drive shaft (322) of the first worm (320). The drive means may include a manual override in case of emergency. Locating bearings (332) or thrust plates position and provide lateral stability for the rotational drive shaft (322) of the first worm (320).

The keel mechanical canting mechanism (220) may include locking mechanisms, the primary arrangement being the self-locking action inherent in a double-enveloping worm gear system. The secondary keel locking arrangement may be formed of an extension to the first worm gear (310) in the form of arc shaped parallel extensions (341, 342) either side of the teeth (311) of the first worm gear (310). The parallel extensions (341, 342) may include multiple apertures (343) spaced radially around the arc of the parallel extensions (341, 342). The secondary keel locking arrangement may include three pins (351, 352, 353) which are in fixed lateral relationship to the sailing vessel's hull, for example by being mounted through two lateral supports (355, 356) of the sailing vessel.

The pins (351, 352, 353) may be driven by a driving component (354) to slide in a longitudinal direction of the pins (351, 352, 353) to be insertable through selected apertures (343) in the parallel extensions (341, 342) to lock the first worm gear (310) at a selected position relative to the sailing vessel's hull.

Referring to Figures 4A to 4E, an embodiment of the mast mechanical canting mechanism (400) is shown. The mast mechanical canting mechanism (400) may correspond to the keel mechanical canting mechanism in an inverted form in the sailing vessel. The mast (150) may be pivoted around an axis of rotation (401) which runs longitudinally fore and aft though the sailing vessel parallel to the axis of rotation (201) of the keel arrangement (120) and vertically above the axis of rotation (201) of the keel arrangement (120).

The mast mechanical canting mechanism (400) may, in a similar arrangement to the keel mechanical canting mechanism (220) include a worm gear (410) and a worm (420), these are referred to as the second worm gear (410) and second worm (420). The second worm gear (410) may rotate in a fixed

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relationship with the mast (150) around the axis (401) of rotation. The second worm (420) may have a helical thread (421) which engages with the teeth (411) of the second worm gear (410).

In one embodiment, the second worm (420) may be a double-enveloping worm for high power transmission which has a waisted-shape which conforms to the arc of the second worm gear (410). It should be understood that the second worm and second worm gear may be recirculating ball worm and worm gear in which the threads are filled with bearing balls that recirculate through the gear and worm as it turns reducing friction and wear in the gear.

The second worm (420) may be driven by a second driving means (430) which may take the form of two motors at either end of a rotational drive shaft (422) of the second worm (420). Locating bearings (432) or thrust plates position and provide lateral stability for the rotational drive shaft (422) of the second worm (420).

The mast mechanical canting mechanism (400) may include locking mechanisms, the

primary mast locking arrangement being similar to the primary locking arrangement for the keel canting. The secondary mast locking arrangement may be formed of an extension to the second worm gear (410) in the form of arc shaped parallel extensions (441, 442) either side of the teeth (411) of the second worm gear (410). The parallel extensions (441, 442) may include multiple apertures (443) spaced radially around the arc of the parallel extensions (441, 442). The secondary mast locking arrangement may include two pins (451, 452) which are in fixed lateral relationship to the sailing vessel's hull. The fixed lateral relationship to the sailing vessel's hull may be provided by mounted holes (457, 458) through two lateral supports (455, 456) of the sailing vessel though which the pins (451, 452) are arranged. The pins (451, 452) may be driven by a driving component (454) to slide in a longitudinal direction of the pins (451, 452) to be insertable through selected apertures (443) in the parallel extensions (441, 442) to lock the second worm gear (410) at a selected position relative to the sailing vessel's hull.

25 Referring to Figure 4E, the mast (150) may include further support in the form of a rigid triangular rigging arrangement formed of two triangular stays (153, 154) and a cross member (463) which support the second worm gear (410). An extended bow sprit (152) may extend along the axis of

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rotation (401) of the mast (150) and form a second triangle with the forestay (151) and the mast (150). A boom (155) is also shown.

Referring to Figure 5A, 5B and 5C which are further views of the keel arrangement (120) shown in Figure 2, further detail of the rotational housing (210) is provided.

The rotational housing (210) may be formed in a generally hollow truncated conical or generally cylindrical form around the shaft (202) of the canting keel arrangement. The keel mechanical canting mechanism (220) of the first worm gear (320) is shown.

The rotational housing (210), rotatable about an axis (201), may have a superior portion (510) of approximately a quadrant of the housing which is located in use within the hull of the sailing vessel.

.0 Two opposing lateral sides (520, 530) form two more approximate quadrants. The opposing lateral sides (520, 530) are of uniform cross-section profile from fore to aft and one of the lateral sides (520, 530) acts as foil promoting lateral resistance

in the water when the keel arrangement is canted and that lateral side (520, 530) rotates below the hull of the sailing vessel. The remaining lower approximate quadrant may be a flat or curved section (540) joining the two keel fins (122, 123) of the keel arrangement (120).

When the keel is in a neutral position (perpendicular to the bottom of the hull) the portion of the rotational housing (210) that extends below the bottom of the hull is generally cylindrical. The inside and outside surfaces are smooth and the thickness fore to aft is uniform, and the walls of the cylinder are parallel to the longitudinal axis of the hull. As the keel cants to windward, the windward side of the cylinder retracts into the hull and the leeward side extends into the water. As the keel continues to cant, the leading edge of the cylinder on the leeward side moves inward toward the longitudinal axis.

Figure 5A shows an embodiment in which the upper portion of the rotational housing (210) is of a generally truncated conical shape.

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In the embodiment of Figure 5A, the inside and outside surfaces are still smooth and the thickness for to aft is still uniform but the walls of the cylinder are no longer parallel to the longitudinal axis. The radius of the aperture at the leading edge is noticeably smaller than the radius of the aperture at the trailing edge giving an overall conical shape, but only in the upper portion of the cylinder i.e. that portion of the cylinder that is in the water when the keel is canted over. When the keel is canted over at 45 degrees, the leading edge on the leeward side may be offset inwards by 5 degrees to the longitudinal axis. The net effect is that when the keel is canted over the leeward side of the housing becomes a semi-circular or semi-conical daggerboard with the leading edge offset inwards by 5 degrees. This will have the effect of countering leeward drift.

.0 Figure 5B shows that the leading edges (521, 531) of the opposing lateral sides (520, 530) are curved from the superior portion (510) towards the aft of the sailing vessel.

Figure 5C shows an alternative embodiment in which the rotational housing (210) is generally cylindrical in shape including the upper portion.

Referring to Figures 6A to 6C, the form of the opposing lateral sides (520, 530) is shown. Figure 6A shows the leading edges (521, 531) of the opposing lateral sides (520, 530) which may have a curved taper.

Figure 6C shows the starboard lateral side (520) rotated externally. The keel arrangement is canted to port by approximately 45 degrees.

The opposing lateral sides (520, 530) may in an alternative embodiment 'foil' towards the inside of the cylinder near to the leading edges (521, 531) with the trailing edges having virtually no foiling of the profile. The flat section (540) may have a constant profile.

Port (or starboard) lateral sides (520, 530) provide a form of quarter-circle foiled daggerboard(s), one retracted the other deployed. Forward motion of the sailing vessel and therefore the lateral side (520, 530) through the water generates "lift" (or force) to counter leeward drift of the hull.

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Leeward drift of the sailing vessel is countered by deployment of the lateral sides (520, 530). When the keel arrangement is canted to windward, the leeward lateral side is deployed whilst the windward lateral side retracts. The forward passage of the lateral side (520, 530) through water generates 'lift', the vector of which opposes the leeward drift of the hull.

- It is envisaged that the rotational housing (210) and lateral sides (520, 530) may be used with (or without) other forms of canting keel including hydraulic canting keels.
  - Referring to Figures 7A to 7F, an embodiment of a ballast bulb (140) is shown. The ballast bulb (140) is provided at the base of the keel arrangement (120) and is of foiled profile in both a longitudinal and lateral cross-section of the ballast bulb (140).
- .0 In the described embodiment, the ballast bulb (140) has two axes of rotation (710, 720) about which the ballast bulb (140) may rotate. Rotation around the first axis of rotation (710) will adjust the roll of the ballast bulb (140) around a hinged support (711) as shown in Figures 7A and 7B. Rotation around the second axis of rotation (720) will adjust the pitch of the ballast bulb (140) around a hinged support (721) as shown in Figures 7C and 7D.
- .5 Figures 7E and 7F show the hinged supports (711, 721) and the first and second axes of rotation (710, 720). The hinged support (721) of the second axis of rotation (720) may be within the ballast bulb (140). The hinged supports (711, 721) may be cylindrical rotational supports.
  - Control of the rotation may be provided by control actuators (818) shown in Figures 8A and 8B provided in the sailing vessel which control four wires (821, 822, 823, 824) extending through the keel arrangement (120) and emerging at the apex (830) of the keel arrangement (120) and then extending externally to the four quadrants of the superior surface (820) of the ballast bulb (140). Tensioning or releasing the four wires (821, 822, 823, 824) by coil winches (811, 812, 813, 814) driven by control actuators (818) may adjust the pitch and roll of the ballast bulb (140).

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The sailing vessel's ballast bulb (140) is configured as a hydrofoil. The hydrofoil may articulate with respect to the keel arrangement via two hinged supports external and internal of hydrofoil casing. External hinged support rotates the hydrofoil around its long axis (in a range of approximately 1400) and the internal hinged support permits varying foil's angle-of-attack (in a range of approximately 200). Rotation around the long axis permits direction of 'lift' generated by foil to be directed between either raising hull relative to water or into a 'righting moment'. Adjustment to the 'angle-of-attack' controls the quantum of 'lift' generated.

Referring to Figure 9A and 9B, an embodiment of a ballast bulb (140) is shown. In this embodiment, the ballast bulb (140) may include a moveable heavy mass (910) within a longitudinal compartment (920) of the ballast bulb (140). Forward or aft adjustment of ballast permits 'trimming' of the vessel's hull.

The moveable heavy mass (910) may be controlled via the actuators (818) shown in Figure 8 with wires (841, 842) extending from coil winches (851, 852) in the sailing vessel. The wires (841, 842) may pass around wheels (861, 862) to slide the moveable heavy mass (910) within the longitudinal compartment (920).

Figure 9A also shows additional compartments (930) which may be provided in the ballast bulb (140) which may house electric batteries for the sailing vessel. Figure 9B shows the ballast bulb (140) in a closed form.

The ballast bulb (140) casing may be demountable into two halves. The ballast bulb (140) may thus be disassembled when the vessel is on 'the hard' thereby accessing internal compartments wherein a heavy metal mass is located. The internal design of housing may be configured into compartments that permit reduction or increase of mass of heavy metal ballast and/or ballast's location. The compartments may additionally permit utilising Valve-Regulated Lead Acid batteries as ballast within the ballast bulb compartments.

25 Referring to Figures 10A and 10B, a retractable propulsion system (1000) may be provided positioned forward of the mast mechanical canting mechanism (400). The propulsion system (1000) may

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comprise a retractable azimuth electric motor powered thruster located amidships forward of amidships of vessel's hull. The propulsion system (1000) may also function as a turbine electric generator.

Referring to Figures 11 A and 11 B, an embodiment of the underside (1100) of the hull (110) is shown including a housing (1110) for the shaft (202) of the keel mechanical canting mechanism (220). The hull may be configured to incorporate a semi-circular recess located in the aft two-thirds of the midline of the hull between port and starboard chines. At least some of the recess houses and permits access to the shaft (202) around which the keel's canting mechanism rotates.

The figures show the hard chines of the hull (110) with a port semi-hull chine (111) and a starboard semi-hull chine (112). A third chine (1120) is shown from the bow of the sailing vessel to the mid-section of the sailing vessel.

Hull construction of a sailing vessel may utilise a hard chine technique. Hard chine reduces cost by

simplifying construction whether in metal, plywood, composites, or other materials. Fabrication of a mould for the hull may be avoided. The proposed arrangement may have three or more bow chines converging aft into two chines in the aft third of the hull configured as port and starboard semi-hulls. The deadrise or 'angle of offset' between port and starboard aft chines is configured such that when leeward semihull's attitude is 'on an even-keel', the windward semi-hull is substantially clear of water. The described design of the hull additionally incorporates features which counter the leeward drift that occurs secondary to the force generated by the wind on the sails. Conventional keeled sailing boats utilise the fin of the keel to counter leeward drift; however, canting the keel increasingly diminishes this counter-force. The hull's sheer strake (leeside of hull) is configured such that when leehull is sailed on an even-keel, the sheer strake is orientated perpendicular to the water surface, thereby imparting a resistance to leeward movement. Additional counter-force to leeward drift is generated by the asymmetry that occurs (port sheer strake relative to starboard sheer strake) as hull

heels to leeward, so windward sheer strake rises clear of the water.

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Vessel's deck and cockpit are configured wherein port and starboard components thereof (offset one to the other) such that when leeward semi-hull is sailing on an even-keel, windward hemi-deck and hemi-cockpit are orientated parallel to water.

The forward termination of the shaft housing (1110) may provide a location for a 3D forward-scanning sonar transducer housing (1130). The housing (1130) may be a bulb located amidships forward of the termination of the shaft housing (1110).

A hatch (1140) for a retractable propulsion system (1000) is shown in the underside (1100) of the hull (110). Slots (1150) are also shown for the location of the lateral sides (520, 530) of the rotational housing (210) of the keel arrangement (120).

.0 Referring to Figures 12A to 12C, an embodiment of the sailing vessel is shown having an integrated water-making apparatus.

Many sailing vessels have water-making apparatus coupled to a diesel engine. The engine is often run for a few hours a day to charge batteries, the water-maker concomitantly generating fresh water. The described water-making apparatus utilises either the helmsman's (standing position) weight or a crew member's (sitting position) weight to provide the power/energy (gravity) required for the reverse osmosis process by which the water-making apparatus functions.

Figure 12A shows a cut away view (1230) of the hull (110) including the lateral supports (455, 456) either side of the mast canting mechanism (400) and the lateral supports (355, 356) on either side of the keel canting mechanism (220). Additional lateral supports or bulkheads are provided at the bow (1231) and at the stern (1232). This figure shows a fresh water storage tank (1234) and a reverse osmosis membrane unit (1235). This figure also shows the watertight enclosure (1233) affixed and sealed to the interior surface of the hull and which surrounds the generally cylindrical or truncated conical housing of the keel arrangement.

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Figure 12B shows an embodiment of a helmsman's weight transmission apparatus (1210) in the form of a hinged (1213) pedestal (1211) which articulates with a reverse-osmosis water generator (1212).

Figure 12C shows an embodiment of a crew member's weight transmission apparatus (1220) in the form of a hinged seat (1221) provided on a portion of the deck (1223) which articulates with a reverseosmosis water generator (1222).

Throughout the specification and claims unless the contents requires otherwise the word 'comprise' or variations such as 'comprises' or 'comprising' will be understood to imply the inclusion of a stated integer or group of integers but not the exclusion of any other integer or group of integers.

Finally, the language used in the specification has been principally selected for readability and instructional purposes, and it may not have been selected to delineate or circumscribe the inventive subject matter. It is therefore intended that the scope of the invention be limited not by this detailed description, but rather by any claims that issue on an application based hereon. Accordingly, the disclosure of the embodiments of the invention is intended to be illustrative, but not limiting, of the scope of the invention, which is set forth in the following claims.

.5 In compliance with the statute, the invention has been described in language more or less specific to structural or methodical features. The term "comprises" and its variations, such as "comprising" and "comprised of" is used throughout in an inclusive sense and not to the exclusion of any additional features.

It is to be understood that the invention is not limited to specific features shown or described since the means herein described comprises preferred forms of putting the invention into effect.

The invention is, therefore, claimed in any of its forms or modifications within the proper scope of the appended claims appropriately interpreted by those skilled in the art.

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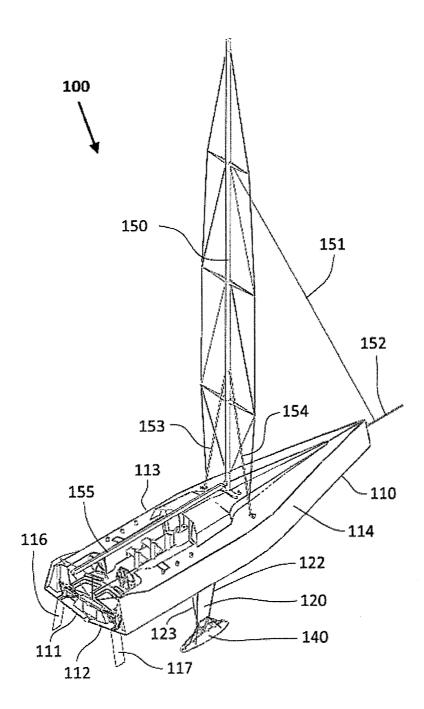
#### **CLAIMS:**

- 1. A sailing vessel having a hull, a keel, a mast, a ballast bulb in the form of a hydrofoil providing lift and ballast at the base of the keel and a control apparatus for controlling adjustment of the ballast bulb in relation to the keel, wherein the control apparatus includes: a first rotating mechanism for rotating the ballast bulb around a lateral axis of the ballast bulb to change the pitch of the ballast bulb in relation to the keel; and a second rotating mechanism for rotating the ballast bulb around a longitudinal axis of the ballast bulb to change the roll of the ballast bulb in relation to the keel, and change the direction of lift generated by the hydrofoil.
- 2. A sailing vessel as claimed in claim 1, wherein said control apparatus includes a pitch actuator and a roll actuator connected to the ballast bulb by means of wires between each quadrant of an upper surface of the ballast bulb and the keel.
- 3. A sailing vessel as claimed in claim 1 or 2, wherein the ballast bulb includes a compartment extending longitudinally within the ballast bulb and a movable mass housed within the compartment and movable longitudinally in the compartment to alter the ballast distribution in the ballast bulb.
- 4. A sailing vessel as claimed in claim 3, wherein the ballast bulb includes at least one further compartment which is accessible when the sailing vessel is out of the water.
  - A sailing vessel having a hull, a keel, a mast, a ballast bulb providing ballast at the base of the keel and a control apparatus for controlling adjustment of the ballast bulb in relation to the keel, wherein the control apparatus includes: a first rotating mechanism for rotating the ballast bulb around a lateral axis of the ballast bulb to change the pitch of the ballast bulb in relation to the keel; and a second rotating mechanism for rotating the ballast bulb around a longitudinal axis of the ballast bulb to change the roll of the ballast bulb in relation to the keel, wherein said control apparatus includes a pitch actuator and a roll actuator connected to the

ballast bulb by means of wires between each quadrant of an upper surface of the ballast bulb and the keel.

- 6. The sailing vessel as claimed in claim 5, wherein the ballast bulb includes a compartment extending longitudinally within the ballast bulb and a movable mass housed within the compartment and movable longitudinally in the compartment to alter the ballast distribution in the ballast bulb.
- 7. The sailing vessel as claimed in claim 6, wherein the ballast bulb includes at least one further compartment which is accessible when the sailing vessel is out of the water.

Fig 1A



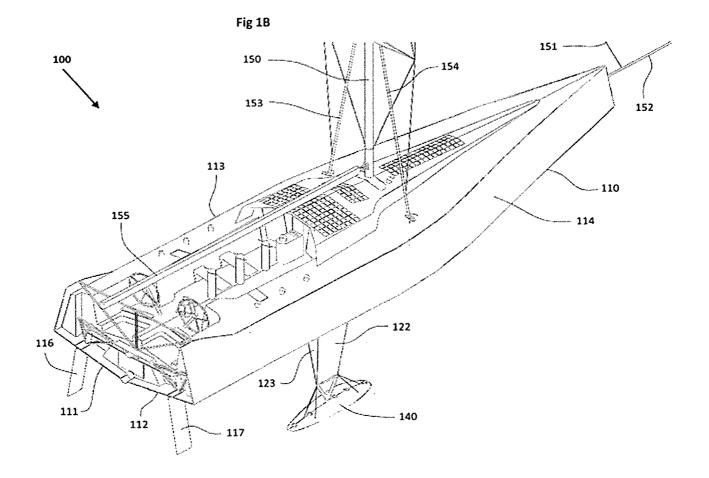
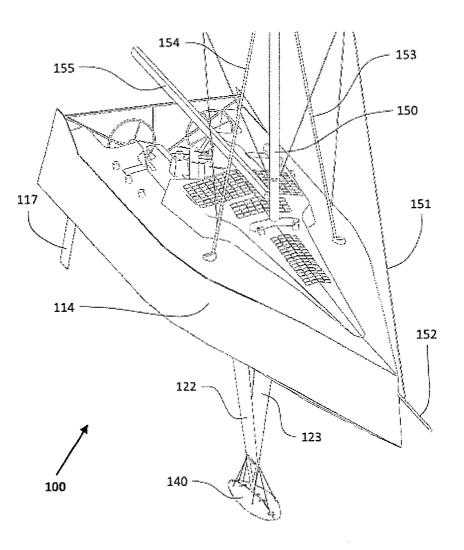
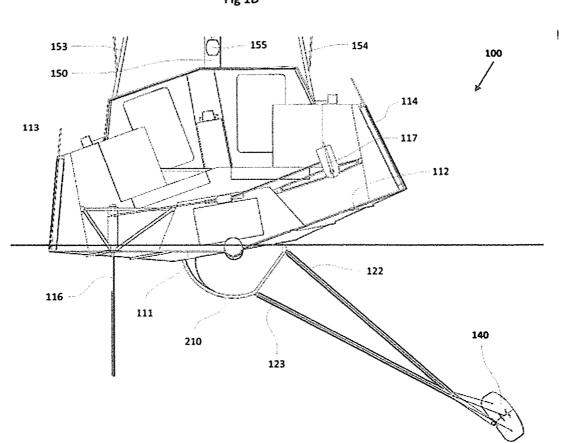
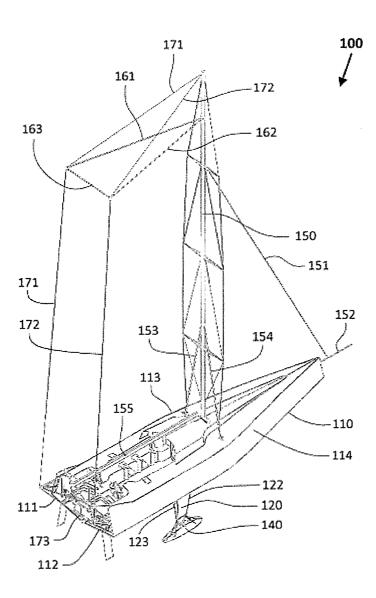


Fig 1C







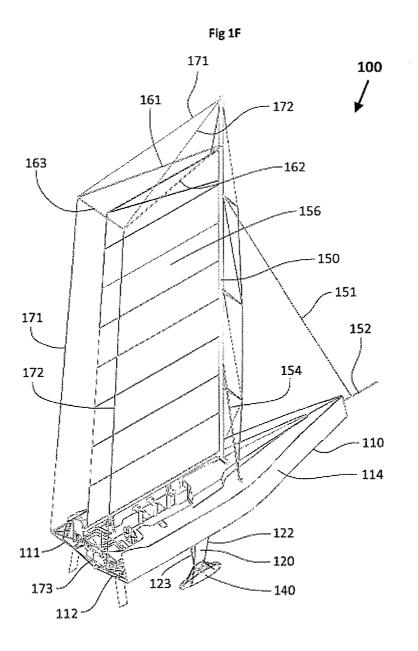


Fig 1G

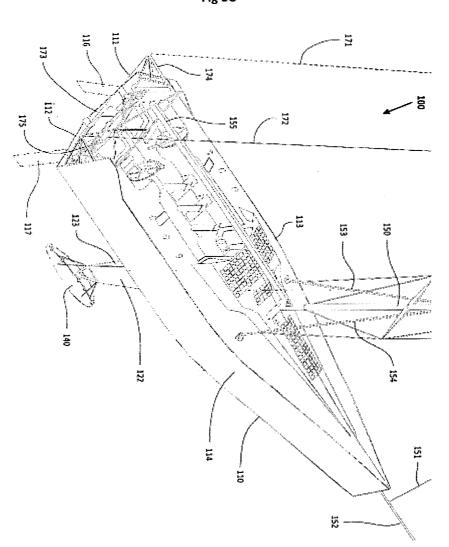


Fig 1H

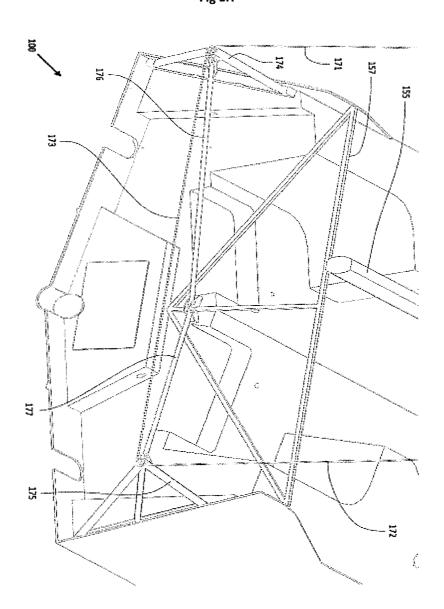


Fig 11

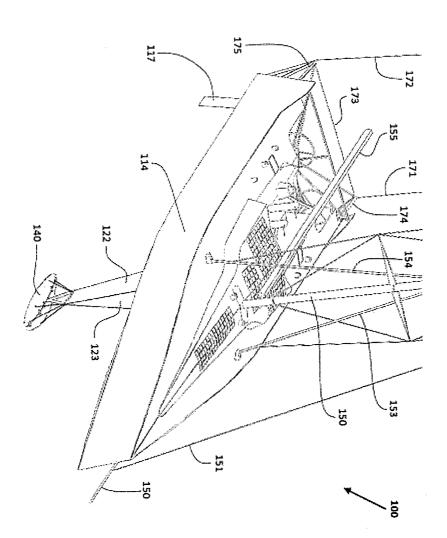
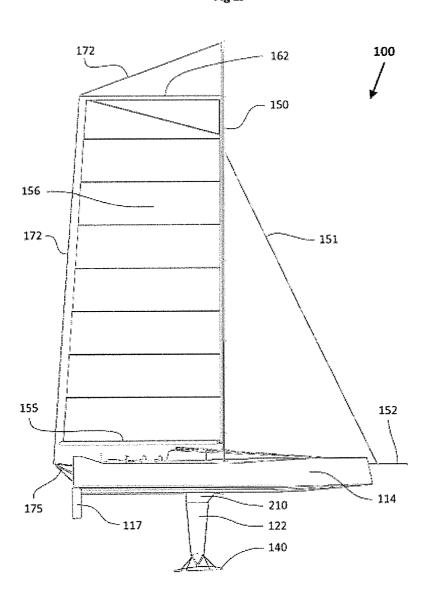


Fig 1J



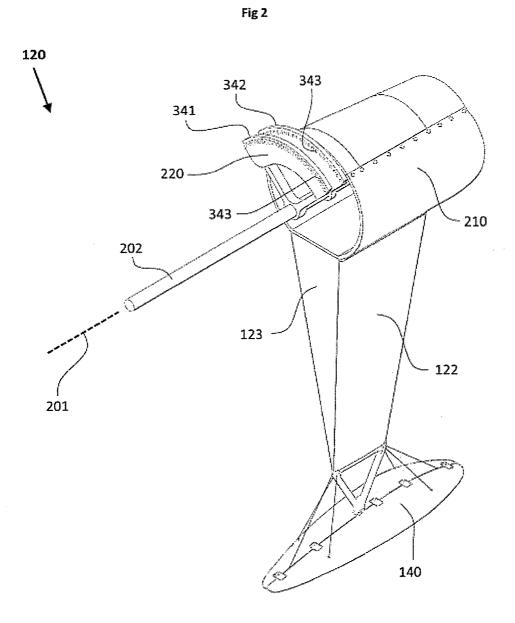


Fig 3A

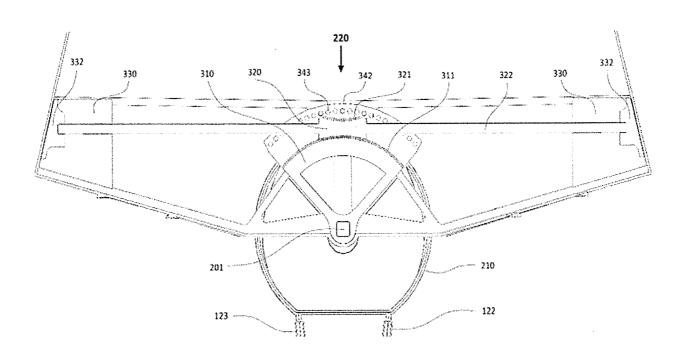


Fig 3B

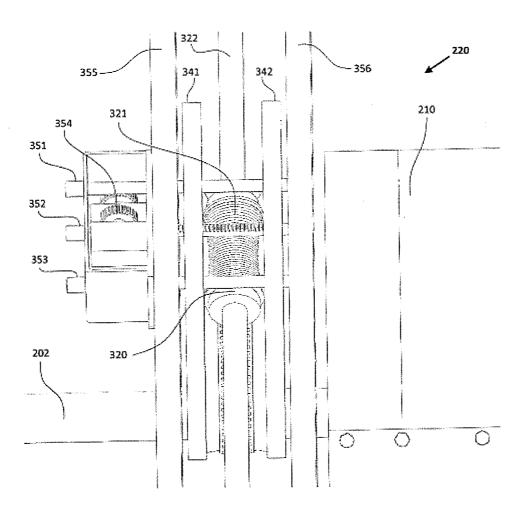
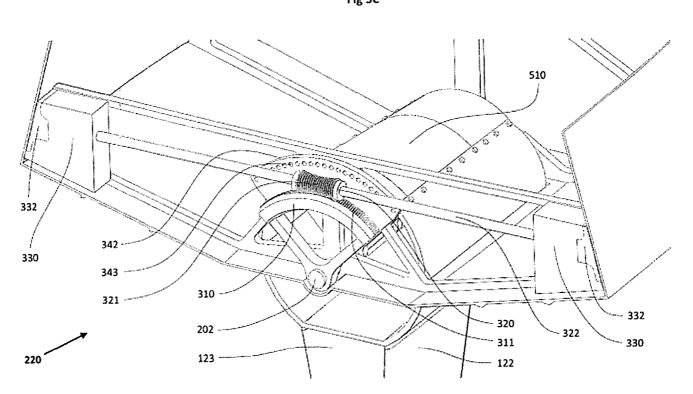


Fig 3C



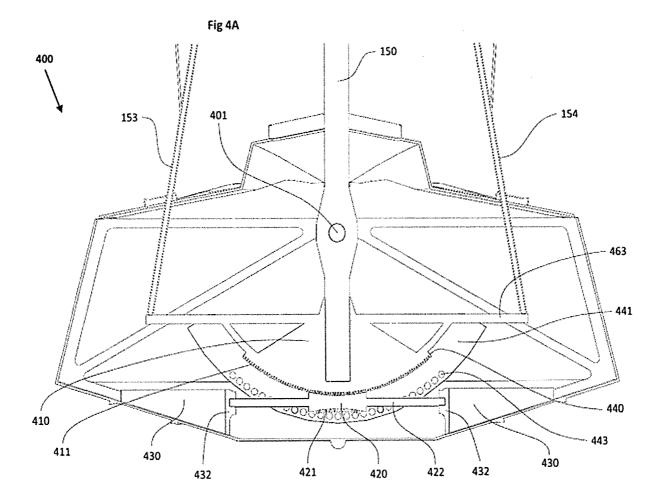
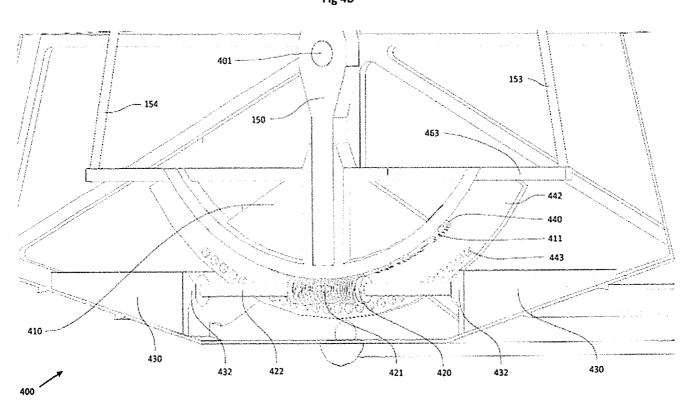
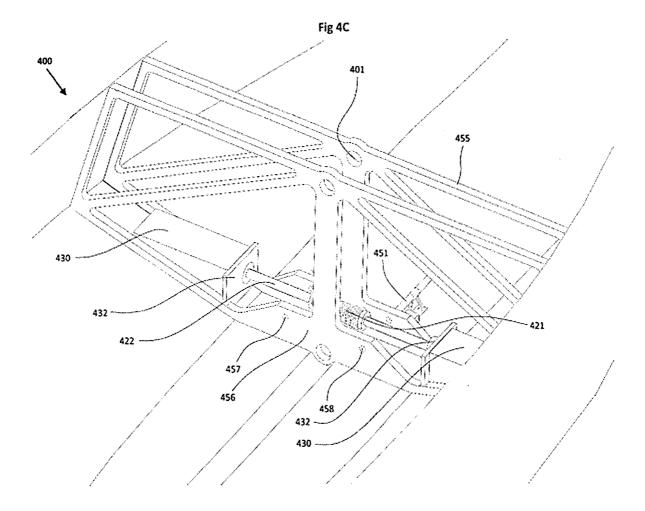
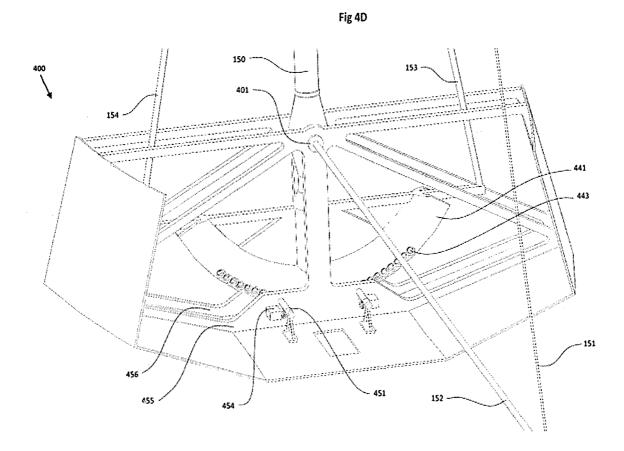


Fig 4B







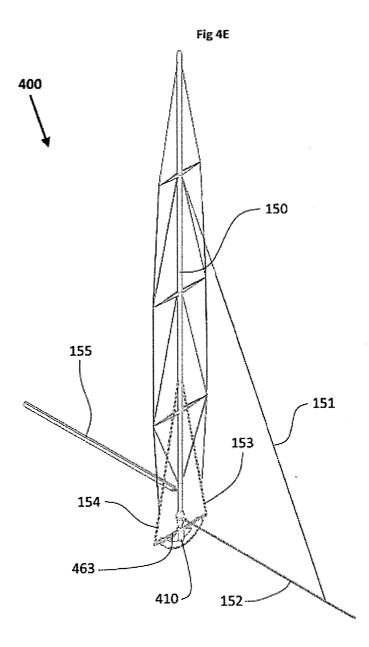


Fig 4F

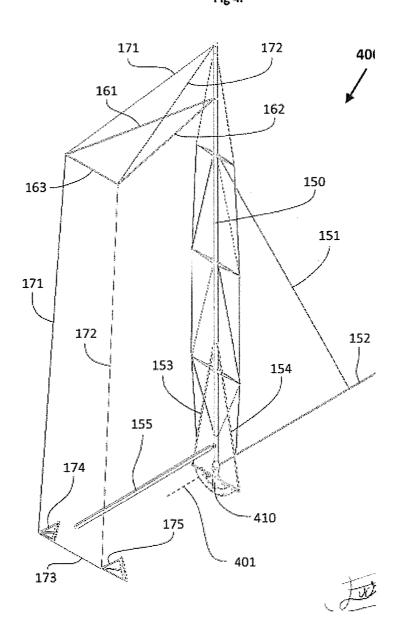


Fig 5A 220 510 201 202 320 530 210 - 540 520 - 123 122 -120 140 -

Fig 5B

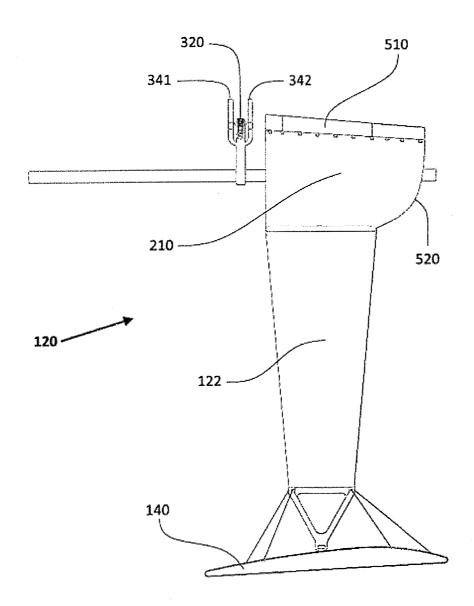
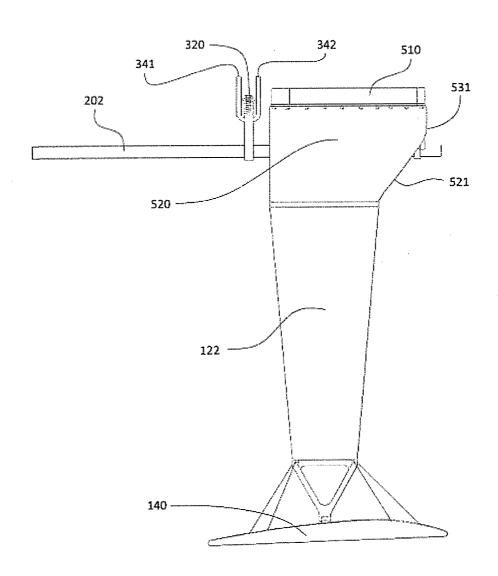


Fig 5C



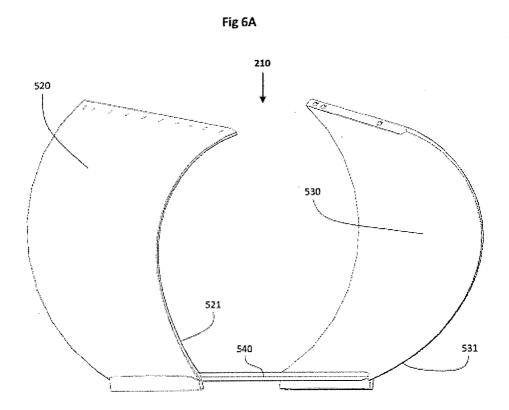


Fig 6B

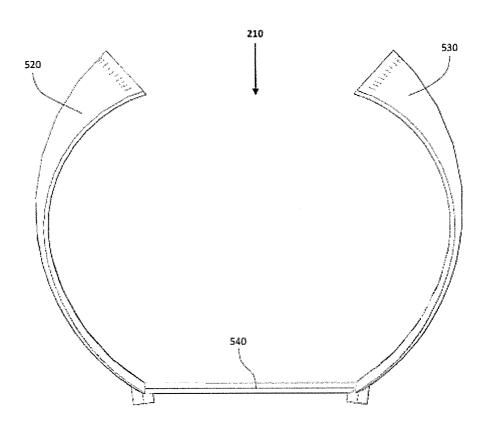
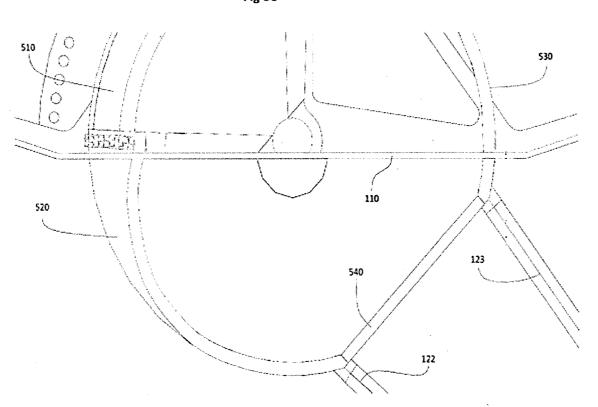


Fig 6C



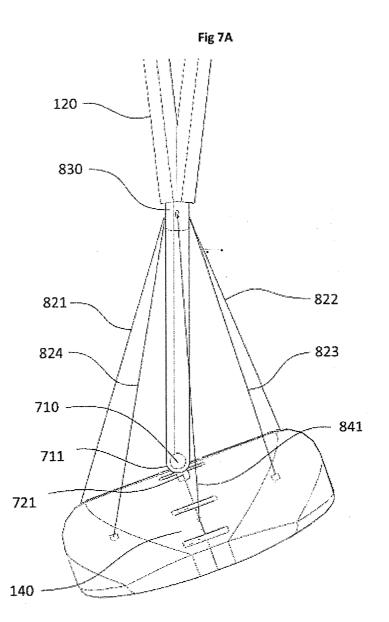
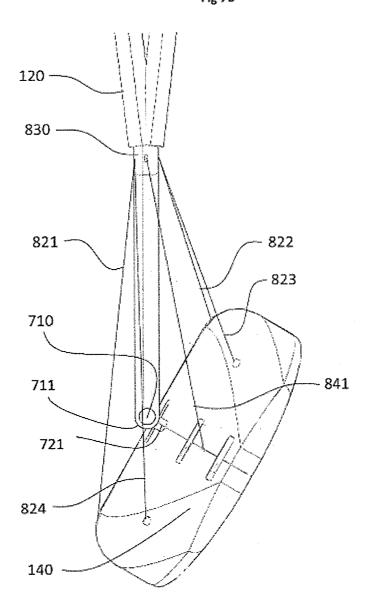
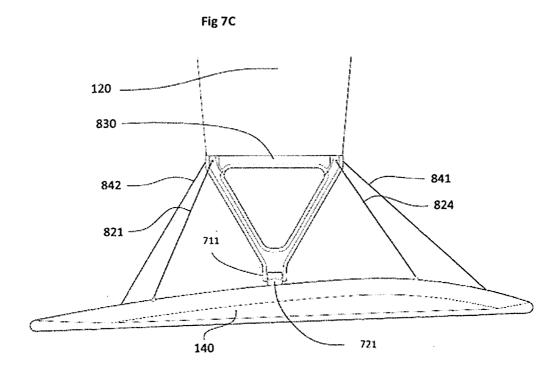
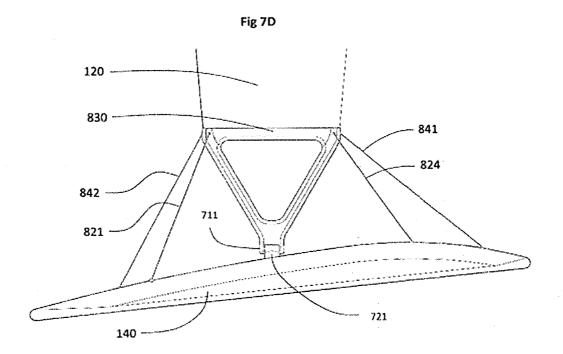


Fig 7B







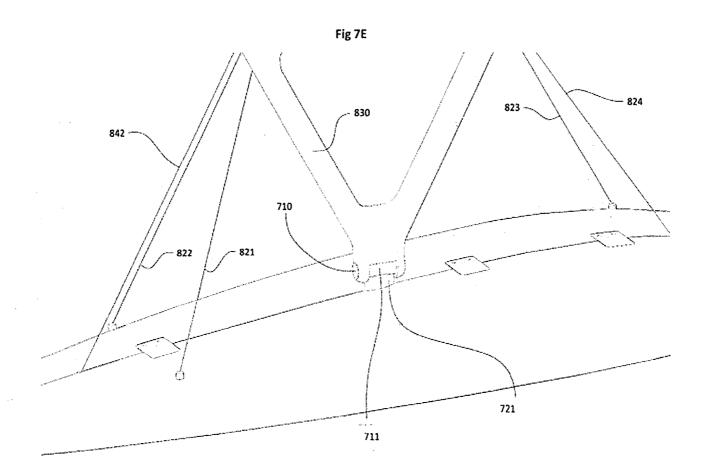


Fig 7F

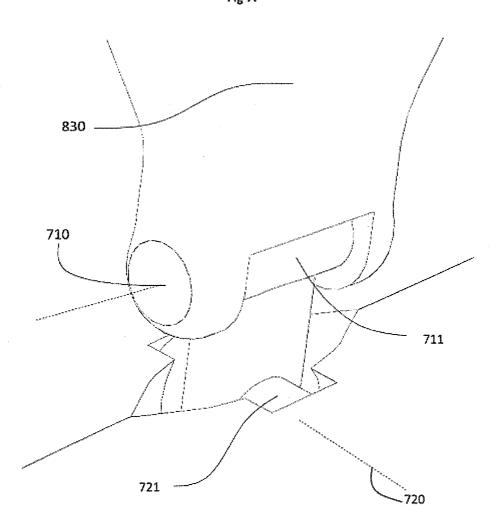
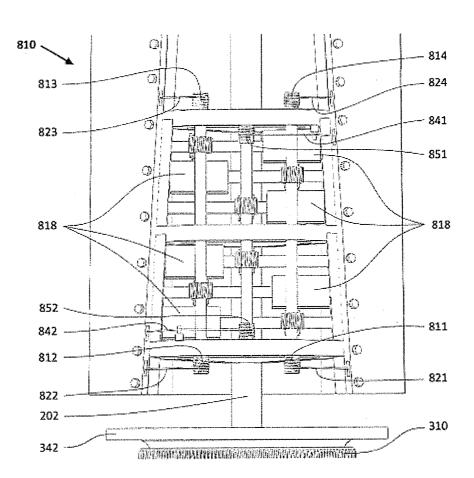


Fig 8A



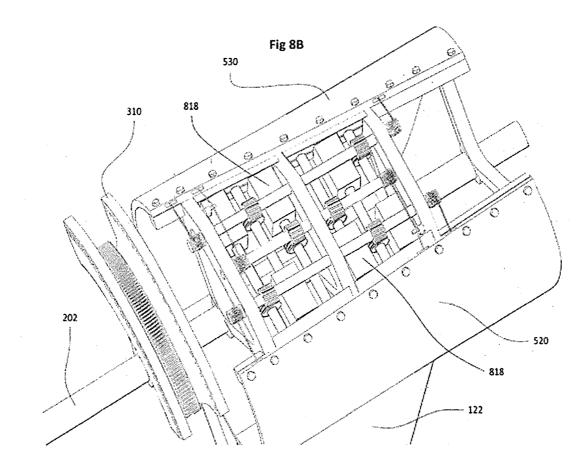


Fig 9A

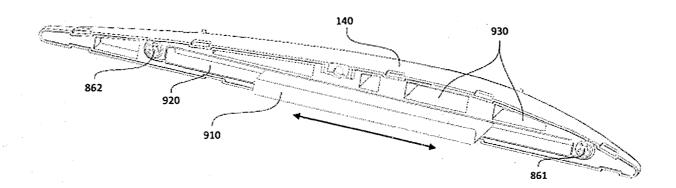
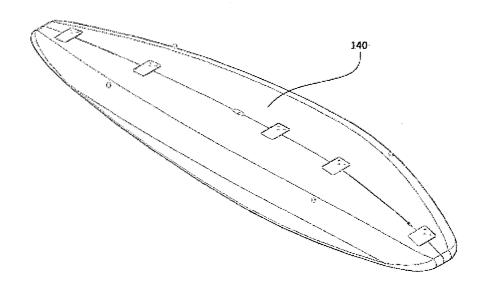


Fig 9B



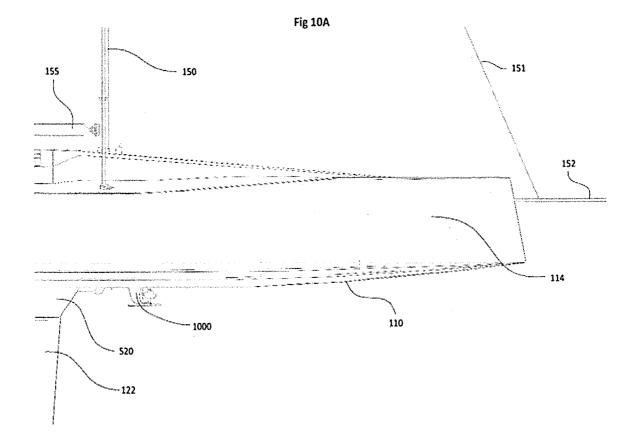


Fig 10B

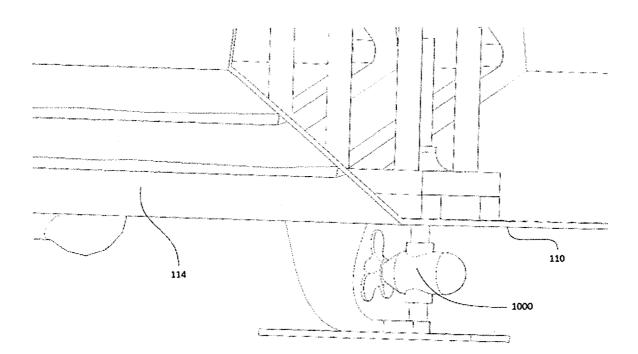


Fig 11A 1100 1120 -1140 1130 -1150 -1110 -112 -- 111

Fig 11B



Fig 12A

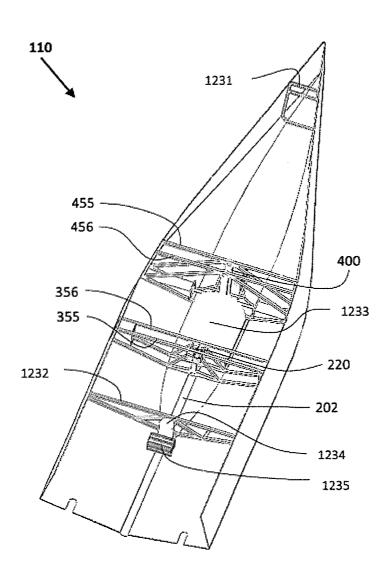


Fig 12B

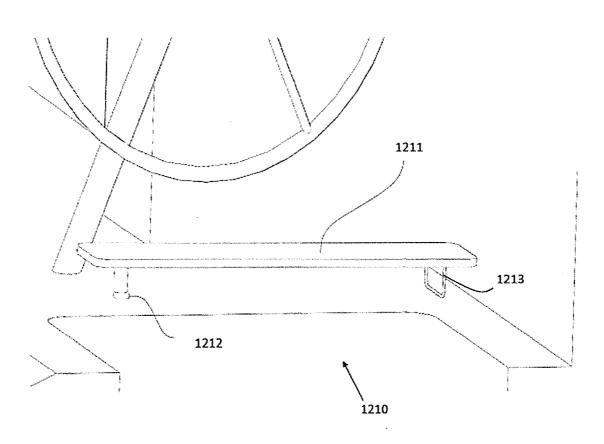


Fig 12C

